

By

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Q → Define different kinds of Elastic Constants.
• Study the relation between elastic constants.
What is the limiting value of Poisson's ratio.

Ans:- Different kinds of elastic constant:-

(i) Young's Modulus of Elasticity γ :-

Young's modulus is defined as the ratio of longitudinal stress to longitudinal strain within the elastic limit. It is usually denoted by the letter γ .

Young's Modulus of Elasticity γ

$$= \frac{\text{longitudinal stress}}{\text{longitudinal strain}}$$

If F be the force applied normally to a cross sectional area 'a', the stress is F/a . And if, there be change l , produced in the original length L the strain is given by l/L . So that

$$\text{Young's Modulus of Elasticity} = \frac{F/a}{l/L} = \frac{F \cdot L}{a \cdot l} \quad \checkmark$$

(ii) Poisson's Ratio ' ν ':-

The ratio of the lateral strain to longitudinal strain is constant for the material and is known as Poisson's ratio. It is

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usually denoted by σ

$$\therefore \text{Poisson's Ratio } \nu = \frac{\text{Lateral strain}}{\text{Longitudinal strain}}$$

If l and r be the initial length and radius, Δl and Δr be the increase and decrease in them respectively, we have

$$\text{Longitudinal strain} = \frac{\Delta l}{l}$$

$$\text{Lateral strain} = \frac{\Delta r}{r}$$

$$\therefore \text{Poisson's Ratio} = \frac{\Delta r/r}{\Delta l/l}$$

$$= \frac{\Delta r}{\Delta l} \frac{l}{r}$$

Poisson's ratio have no dimension

(iii) Bulk modulus of elasticity K :-

It is ratio of the uniform bulk stress to bulk strain. It is usually denoted by the letter K

\therefore Bulk modulus of elasticity K

$$= \frac{\text{Bulk stress}}{\text{Bulk strain}}$$

Thus, if F be the force applied uniformly and normally on the surface area a , the stress is F/a and if ΔV be the change in volume produced in original volume V

the strain is $\frac{\theta}{V}$ and therefore

Bulk modulus of elasticity K

$$= \frac{F/a}{\theta/V}$$

$$= \frac{F}{a} \cdot \frac{V}{\theta}$$

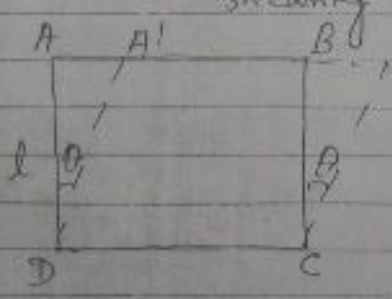
$$= \frac{PV}{\theta} \quad \left[\because P = \frac{F}{a} \right]$$

(iv) Modulus of rigidity η :-

The ratio of the shearing stress to the shearing strain is called the modulus of rigidity. It is usually represented by the letter η (eta).

Modulus of rigidity η

$$= \frac{\text{shearing stress}}{\text{shearing strain}}$$



If a be the area of the upper surface and θ be the angle through which the vertical

side AD and BC are turned, we have tangential

$$\text{shearing stress} = \frac{F}{a}$$

$$\text{shearing strain} = \theta = \frac{x}{l}$$

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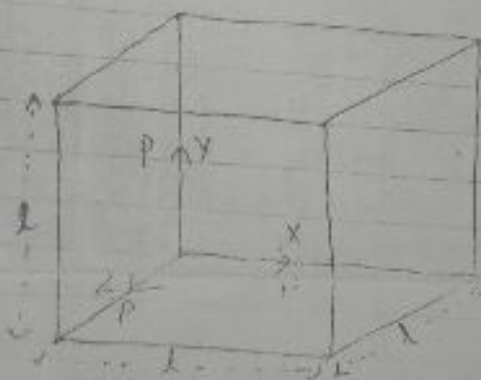
∴ Modulus of rigidity.

$$\eta = \frac{F/a}{\theta}$$

$$= \frac{F/a}{\frac{x}{l}}$$

$$= \frac{F}{a} \cdot \frac{l}{x}$$

Relation between different elastic constants of a substance :-



To deduce the necessary relation between the different elastic constants of a substance. We consider a unit cube of the substance which is subjected to a bulk stress P i.e. the same stress (P) acts along the three axes (Ox , Oy , Oz) of the cube simultaneously. Since the original dimension of the substance is unity so the change in dimension will be equal to the strain produced.

From definition, γ

$$= \frac{\text{Longitudinal stress } (P)}{\text{Longitudinal strain}}$$

$$\therefore \text{Longitudinal strain} = + \frac{P}{Y}$$

\therefore Poisson's ratio (ν)

$$= \frac{\text{Lateral strain}}{\text{Longitudinal strain}}$$

$$\therefore \text{Lateral strain} = \nu \times \text{Longitudinal strain}$$

$$= -\nu \cdot \frac{P}{Y}$$

The elongation is taken to be positive whereas contraction is taken negative.

In order to calculate the strain produced in the cube by the application of the bulk stress ($+P$), let us suppose that the stress ($+P$) acts along the three axes in succession and the corresponding strains produced along the three axes are represented in a tabular form below.

stress applied along strain produced along

stress applied along			strain produced along		
OX	OY	OZ	OX	OY	OZ
+P	0	0	$+\frac{P}{Y}$	$-\nu \frac{P}{Y}$	$-\nu \frac{P}{Y}$
0	+P	0	$-\nu \frac{P}{Y}$	$+\frac{P}{Y}$	$-\nu \frac{P}{Y}$
0	0	+P	$-\nu \frac{P}{Y}$	$-\nu \frac{P}{Y}$	$+\frac{P}{Y}$
$\frac{P}{3}$	$\frac{P}{3}$	$\frac{P}{3}$	$\frac{P}{Y}(1-\nu)$	$\frac{P}{Y}(1-\nu)$	$\frac{P}{Y}(1-\nu)$

or stress $\frac{P}{3}$ vertically

Thus under the action of the bulk stress (+P) each side of the cube is elongated in length by $\frac{P}{Y}(1-2\sigma)$.

\therefore length of each side of the elongated cube $= 1 + \frac{P}{Y}(1-2\sigma)$

$$\begin{aligned} \therefore \text{Volume of the deformed cube} &= \left\{ 1 + \frac{P}{Y}(1-2\sigma) \right\}^3 \\ &= 1 + \frac{3P}{Y}(1-2\sigma) + \dots \\ &= 1 + \frac{3P}{Y}(1-2\sigma) \end{aligned}$$

$$\begin{aligned} \therefore \text{change in volume of the cube} &= \text{Bulk strain produced} \\ &= 1 + \frac{3P}{Y}(1-2\sigma) - 1 \\ &= \frac{3P}{Y}(1-2\sigma) \end{aligned}$$

\therefore Bulk modulus of elasticity K

$$= \frac{\text{Bulk stress}}{\text{Bulk strain}}$$

$$= \frac{P}{\frac{3P}{Y}(1-2\sigma)}$$

$$= \frac{Y}{3(1-2\sigma)}$$

$$\therefore Y = 3K(1-2\sigma) \dots \dots \textcircled{1}$$

Let us now suppose that the cube is subjected to a shearing

stress P which is equivalent to a stress $+P$ and a stress $-P$ acting at right angles the corresponding strain produced are shown below in the table.

	OX	OY	OZ	OX	OY	OZ
	$+P$	0	0	$+P/\gamma$	$-P/\gamma$	$-P/\gamma$
	0	$-P$	0	$+P/\gamma$	$-P/\gamma$	$+P/\gamma$
On OZ direction Vertically		$-P$	0	$+P(1+\sigma)/\gamma$	$-P(1+\sigma)/\gamma$	0

$$\therefore \text{Shearing strain} = \text{Elongational strain} + \text{Contractional strain}$$

$$= \frac{P}{\gamma}(1+\sigma) + \frac{P}{\gamma}(1+\sigma)$$

$$= \frac{2P}{\gamma}(1+\sigma)$$

\therefore Modulus of rigidity

$$\eta = \frac{\text{shearing stress}}{\text{shearing strain}}$$

$$\text{By Prof. Shailendra Kumar} = \frac{P}{\frac{2P}{\gamma}(1+\sigma)}$$

$$6205229503 = \frac{\gamma}{2(1+\sigma)}$$

$$\therefore \gamma = 2\eta(1+\sigma) \dots \dots \dots (ii)$$

$$\therefore 3K(1-2\sigma) = 2\eta(1+\sigma) = \gamma \dots \dots (iii)$$

Limiting value of Poisson's ratio's

We have proved above that

$$3K(1-2\sigma) = 2\eta(1+\sigma) = \gamma$$

where K and η are essentially positive quantity.

In the limit $\gamma = 0$. [since body can not deform
 $\therefore \gamma = 0$]

$$\therefore 3K(1-2\sigma) = 0 \text{ or } 2\eta(1+\sigma) = 0$$

$$\therefore (1-2\sigma) = 0 \text{ (}\because 3K \neq 0\text{) or } (1+\sigma) = 0$$

$\sigma = -1$

$$\text{or } 2\sigma = 1 \quad \text{or } \sigma = -1$$

$$\text{or } \sigma = +\frac{1}{2} \quad \text{or } \sigma = -1$$

\therefore The limiting value of σ lie between -1 to $+\frac{1}{2}$.

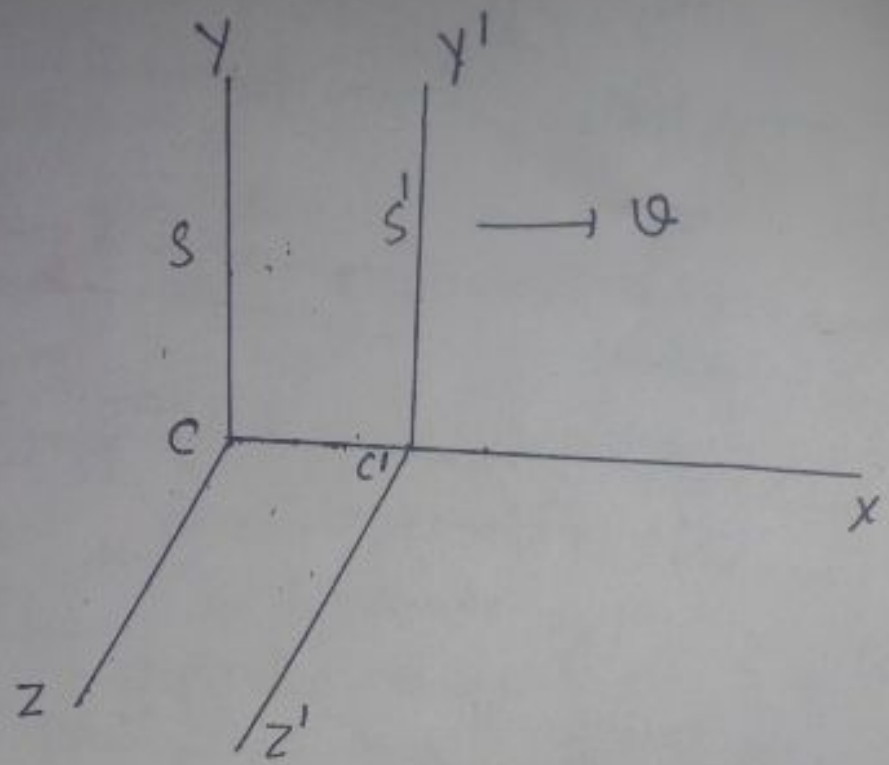
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Q: Derive Lorentz's Transformation Equation.

Ans: → Lorentz's Transformation Equations:

Suppose that S and S' are two inertial frames. S is at rest and S' is moving with a constant velocity v along the common x -axis. To simplify the problem we assume that the x -axis of both the frames is along the direction of travel of the primed frame (S'). As there is no motion along y -axis and z -axis, y & z co-ordinate will transform unchanged i.e. $y = y'$, $z = z'$. but x must transform linearly into x' and t must transform linearly into x' & t' .

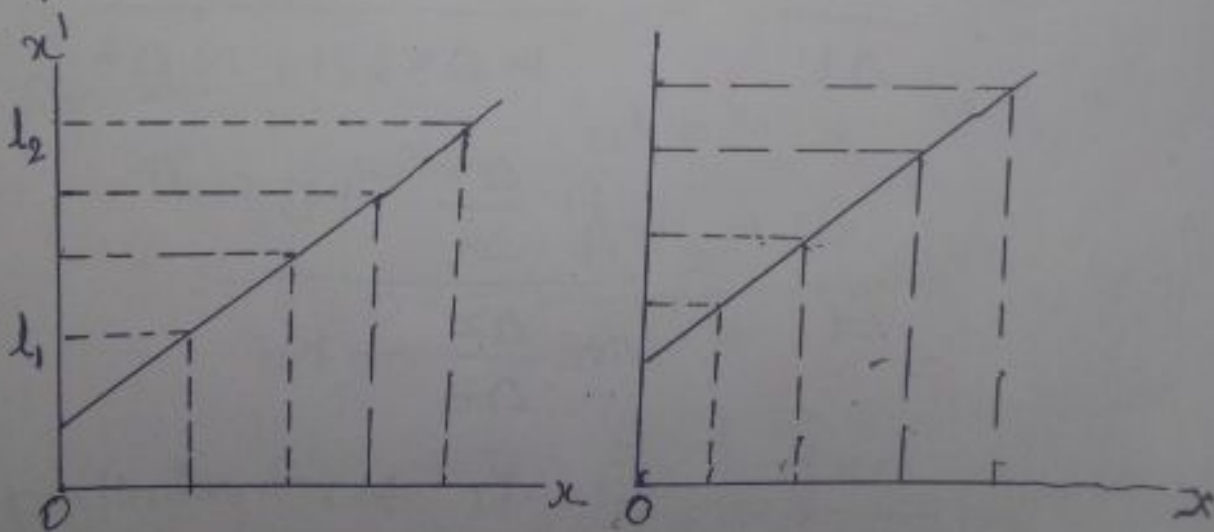
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This is needed for homogeneity of space & time. When the transformation is linear, the length of a rod will not depend on what region of space it is located. We measure the length or distance by comparing with a 'measuring rod', so it is necessary that if $l_1 = l_2$ where l_1 is the length of the rod and l_2 is the reading on the meter scale in one frame (~~the~~ unprimed frame), then in the primed frame also $l_1' = l_2'$. So that the rod and scale

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Contract or elongate equally and an observer get the same reading on the scale. If the transformation is nonlinear then it follows that if $l_1 = l_2$, $l_1' \neq l_2'$ i.e. the length of the rod would depend upon the region of space where it is located. This is against the principle of homogeneity of space.



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A similar argument can be made for time. Therefore, we seek for relativistic transformations in forms

$$x' = Ax + Bt$$

$$t' = Mx + Nt$$

where A, B, M & N are constants to be determined.

$$\therefore \Delta x' = A\Delta x + B\Delta t$$

$$\Delta t' = M\Delta x + N\Delta t$$

$$\therefore \frac{\Delta x'}{\Delta t'} = \frac{A\Delta x + B\Delta t}{M\Delta x + N\Delta t}$$

$$= \frac{A \frac{\Delta x}{\Delta t} + B}{M \frac{\Delta x}{\Delta t} + N}$$

Now $\frac{\Delta x'}{\Delta t'}$ is the velocity of a primed frame.

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$\frac{\Delta x}{\Delta t} = u$, is the velocity of a particle in unprimed frame.

$$\therefore u' = \frac{Au + B}{Mu + N}$$

This relation must hold good for any velocity of the particle in the primed frame and hence must hold for $u' = 0$ as well. But when $u' = 0$, the particle is at rest in the primed frame and has velocity u relative to the unprimed frame (S).

Thus when $u' = 0$, $u = v$

$$\therefore 0 = \frac{Av + B}{Mv + N}$$

$$\therefore Av + B = (Mv + N) \times 0$$
$$= 0$$

$$\therefore B = -Av$$

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$$\therefore u' = \frac{A(u-v)}{Mu+N}$$

where now A , M & N are only unknowns to be determined. This relation must also hold good when the particle is at rest in the frames. Thus $u=0$ and $u' = -v$

$$\therefore -v = \frac{A(0-v)}{N}$$

$$\therefore A = N$$

$$\therefore u' = \frac{A(u-v)}{Mu+N}$$

where now A & M are only two unknowns to be determined.

on the basis of the ~~principle~~ principle of constancy of velocity of light in a vacuum in all

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Inertial frames, we have

$$u = u' = c$$

$$\therefore c = \frac{A(c - v)}{mc + A}$$

$$\therefore m = -A \frac{v}{c^2}$$

Putting the value of the constant B , m and N in terms of A obtained above, we have transformation equation

$$\begin{aligned} x' &= Ax - Avt \\ &= A(x - vt) \end{aligned}$$

$$\begin{aligned} \text{and } t' &= -A \frac{v}{c^2} x + At \\ &= A \left(t - \frac{vx}{c^2} \right) \end{aligned}$$

where A is the only one constant left that is to be determined

According to first postulate of the theory of relativity i.e.

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equivalence of all inertial frames, the above relations must hold good if S' is at rest and S moves with velocity $-v$.

Thus we must have

$$x = A(x' + vt')$$

$$\text{and } t = A\left(t' + \frac{vx'}{c^2}\right)$$

Substituting the values of x and t , we have

$$x' = A^2 \left\{ x' + vt' - v\left(t' + \frac{vx'}{c^2}\right) \right\}$$

$$\text{or } x' = A^2 x' \left(1 - \frac{v^2}{c^2}\right)$$

$$\text{or } A = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}}$$

Thus finally we have transformed equations as

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$$x' = \frac{x - vt}{\sqrt{1 - \frac{v^2}{c^2}}}, \quad y = y', \quad z = z'$$

$$t' = \frac{t - vx}{\sqrt{1 - \frac{v^2}{c^2}}}$$

These are called Lorentz transform equations.

Replacing primed quantities by the corresponding unprimed quantities and unprimed quantities by the corresponding primed one ~~one~~ and v by $-v$, we obtain

$$x = \frac{x' + vt'}{\sqrt{1 - \frac{v^2}{c^2}}}, \quad y = y', \quad z = z'$$

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$$t = \frac{t' + \frac{vx'}{c^2}}{\sqrt{1 - \frac{v^2}{c^2}}}$$

These are called reciprocal transformation equations.

Q:- Write short notes on :- (a) gradient, (b) divergence and (c) curl.

Ans: Gradient:-

The gradient of a scalar point function ϕ is defined as $\nabla\phi$ and is written as $\text{grad } \phi$.

$$\begin{aligned}\text{grad } \phi &= \nabla\phi = \left(i \frac{\partial}{\partial x} + j \frac{\partial}{\partial y} + k \frac{\partial}{\partial z} \right) \phi \\ &= i \frac{\partial \phi}{\partial x} + j \frac{\partial \phi}{\partial y} + k \frac{\partial \phi}{\partial z}\end{aligned}$$

$\text{grad } \phi$ is a vector quantity.

$\phi(x, y, z)$ is a function of three independent variables and its total differential $d\phi$ is

$$\begin{aligned}d\phi &= \frac{\partial \phi}{\partial x} dx + \frac{\partial \phi}{\partial y} dy + \frac{\partial \phi}{\partial z} dz \\ &= \left(i \frac{\partial \phi}{\partial x} + j \frac{\partial \phi}{\partial y} + k \frac{\partial \phi}{\partial z} \right) \cdot (i dx + j dy + k dz)\end{aligned}$$

[$\vec{r} = xi + yj + zk$
 $d\vec{r} = i dx + j dy + k dz$]

$$= (\nabla\phi) \cdot d\vec{r}$$

$$= |\nabla\phi| |d\vec{r}| \cos\theta$$

The value of $d\phi$ is the greatest when $\theta = 0$. It is this property of ϕ that gives its name, the gradient of ϕ .

① Divergence: \rightarrow

The divergence of a vector point function \vec{F} is denoted by $\text{div } \vec{F}$ and is defined as below.

$$\text{Let } \vec{F} = F_1 \hat{i} + F_2 \hat{j} + F_3 \hat{k}$$

$$\text{div } \vec{F} = \nabla \cdot \vec{F}$$

$$= \left(\hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z} \right) \cdot (F_1 \hat{i} + F_2 \hat{j} + F_3 \hat{k})$$

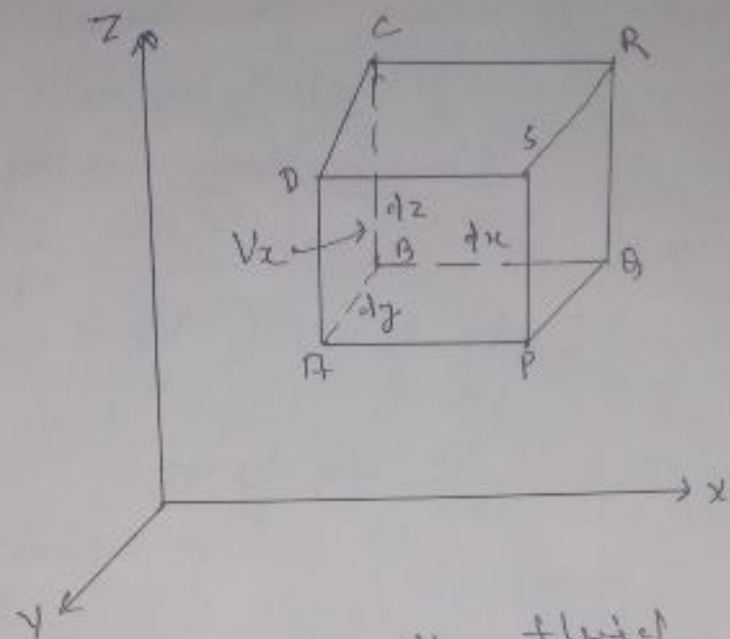
$$= \frac{\partial F_1}{\partial x} + \frac{\partial F_2}{\partial y} + \frac{\partial F_3}{\partial z}$$

It is evident that $\text{div } \vec{F}$ is a scalar function.

Physical Interpretation of Divergence: \rightarrow

Let us consider the case of a fluid flow. Consider a small rectangular parallelepiped of dimensions dx , dy , dz parallel to x , y and z axes respectively.

$$\text{Let } \vec{V} = V_x \hat{i} + V_y \hat{j} + V_z \hat{k}$$



28) be the velocity of the fluid at $P(x, y, z)$.

\therefore Mass of fluid flowing in through the face ABCD per time
 $=$ velocity \times Area of the face
 $= V_z (dy dz)$

Mass the fluid flowing out across the face PQRS per unit time
 $= V_x (x + dx) (dy dz)$
 $= \left(V_x + \frac{\partial V_x}{\partial x} dx \right) (dy dz)$

Net decrease in mass of fluid in the parallelepiped corresponding to the flow along x-axis per unit time

$$= V_x \, dx \, dy \, dz - \left(V_x + \frac{\partial V_x}{\partial x} dx \right) dx \, dy \, dz$$

$$= - \frac{\partial V_x}{\partial x} dx \, dy \, dz$$

(Minus sign shows decrease)

Similarly, the decrease in mass of fluid to the flow along y-axis

$$= \frac{\partial V_y}{\partial y} dx \, dy \, dz$$

and the decrease in mass of fluid to the flow along z-axis

$$= \frac{\partial V_z}{\partial z} dx \, dy \, dz$$

Total decrease of the amount of fluid per unit time

$$= \left(\frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} + \frac{\partial V_z}{\partial z} \right) dx \, dy \, dz$$

Thus the rate of loss of fluid per unit volume

$$= \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z}$$

$$= \left(i \frac{\partial}{\partial x} + j \frac{\partial}{\partial y} + k \frac{\partial}{\partial z} \right) \cdot (i v_x + j v_y + k v_z)$$

$$= \nabla \cdot \vec{v} = \text{div } \vec{v}$$

If the fluid is incompressible, there can be no gain or loss in the volume element. Hence

$$\text{div } \vec{v} = 0 \quad \text{--- (1)}$$

and \vec{v} is called a solenoidal vector function.

Equation (1) is also called the equation of continuity.

(c) Curl :- The curl of a vector point function

\vec{F} is defined as below:

$$\text{Curl } \vec{F} = \nabla \times \vec{F} \quad \text{where } \vec{F} = F_1 i + F_2 j + F_3 k$$

$$= \left(i \frac{\partial}{\partial x} + j \frac{\partial}{\partial y} + k \frac{\partial}{\partial z} \right) \times (F_1 i + F_2 j + F_3 k)$$

$$= \begin{vmatrix} i & j & k \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ F_1 & F_2 & F_3 \end{vmatrix}$$

$$= i \left(\frac{\partial F_3}{\partial y} - \frac{\partial F_2}{\partial z} \right) - j \left(\frac{\partial F_3}{\partial x} - \frac{\partial F_1}{\partial z} \right) + k \left(\frac{\partial F_2}{\partial x} - \frac{\partial F_1}{\partial y} \right)$$

physical meaning of curl:-

We know $V = \omega \times \vec{r}$ where ω is the angular velocity, V is the linear velocity and \vec{r} is the position vector of a point on the rotating body.

$$\begin{aligned} \text{Curl } V &= \nabla \times V \\ &= \nabla \times (\omega \times \vec{r}) \\ &= \nabla \times [(\omega_1 i + \omega_2 j + \omega_3 k) \times (x i + y j + z k)] \end{aligned}$$

$$10 \quad \Delta \times \begin{vmatrix} i & j & k \\ \omega_1 & \omega_2 & \omega_3 \\ x & y & z \end{vmatrix}$$

$$= \Delta \times \left[(\omega_2 z - \omega_3 y) i - (\omega_1 z - \omega_3 x) j + (\omega_1 y - \omega_2 x) k \right]$$

$$= \left(i \frac{\partial}{\partial x} + j \frac{\partial}{\partial y} + k \frac{\partial}{\partial z} \right) \times \left[(\omega_2 z - \omega_3 y) i - (\omega_1 z - \omega_3 x) j + (\omega_1 y - \omega_2 x) k \right]$$

$$11 \quad \begin{vmatrix} i & j & k \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ \omega_2 z - \omega_3 y & \omega_3 x - \omega_1 z & \omega_1 y - \omega_2 x \end{vmatrix}$$

$$= (\omega_1 + \omega_2) i - (-\omega_2 - \omega_2) j + (\omega_3 + \omega_3) k$$

$$= 2(\omega_1 i + \omega_2 j + \omega_3 k) = 2\omega$$

$\text{curl } v = 2\omega$ which shows that
curl of a vector field is
connected with rotational properties
of the vector field and justifies
the name rotation used for curl.

If $\text{curl } \vec{F} = 0$, the field F
is termed irrotational.

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